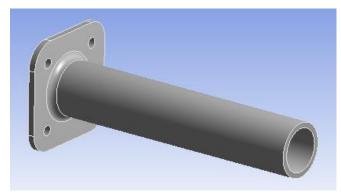


Effect of End Fixity Modeling on Vibration Behavior of Bolted Components

Background Information

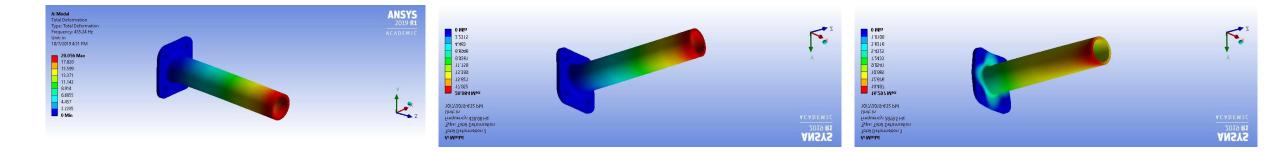
- Boundary conditions for bolted joints are usually modelled as fixed constraints at bolt holes.
- Three approaches are discussed for modeling bolted joints:
 - 1. Fixed constraints at bolt holes
 - 2. Explicit modeling of bolts and attachment
 - 3. Fixing remote points associated with bolt holes
- Component simplified as cantilevered cylinder with end flange attached with four bolts

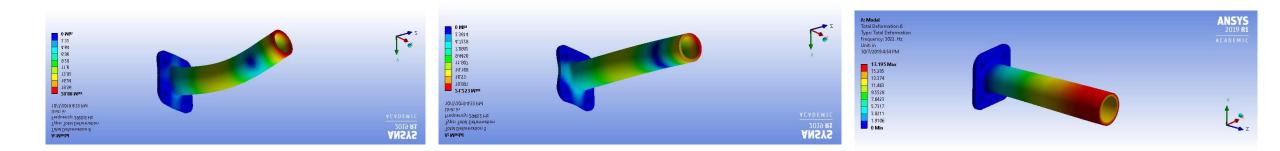


• Modal and random vibration results are compared for the three approaches



Modal Analysis of Component with Fixed Constraints at Bolt Holes

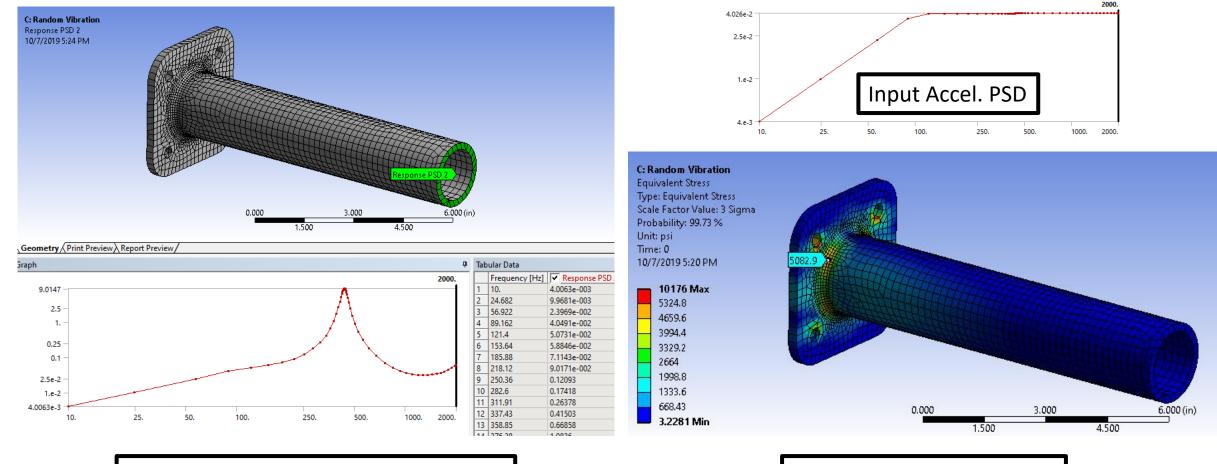




First Six Modes with Holes Fixed



Acceleration & Stress Response to Random Vibration

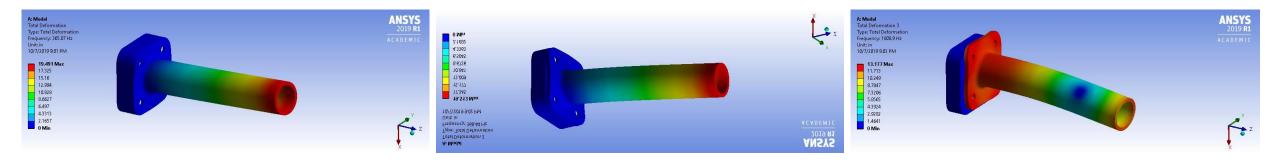


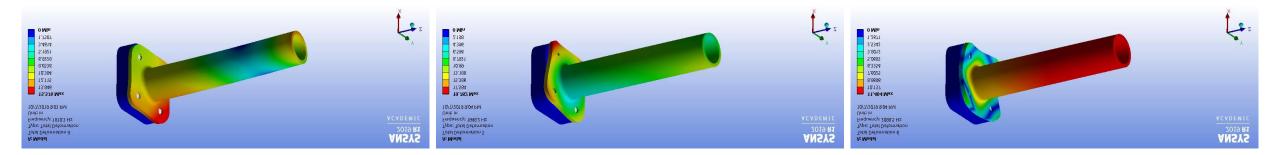
Max. Acc. PSD Response = $9.01 \text{ G}^2/\text{Hz}$

Max. Equiv. Stress = 5.1 KSI



Modal Analysis of Component with Explicit Bolts & Fixture

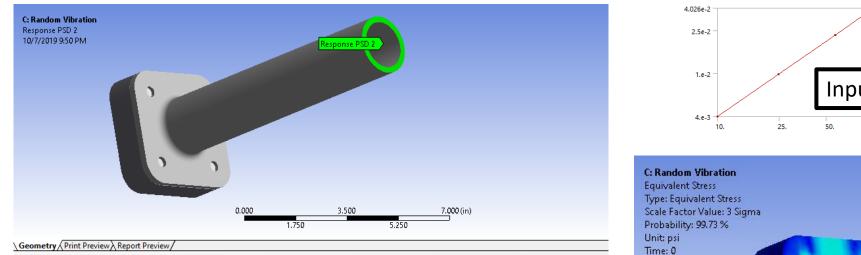


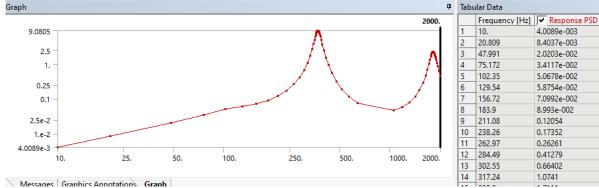


First Six Modes with Explicit Bolts & Fixture

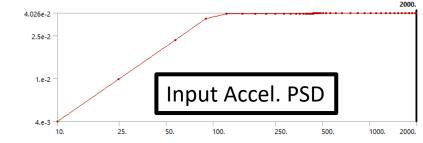


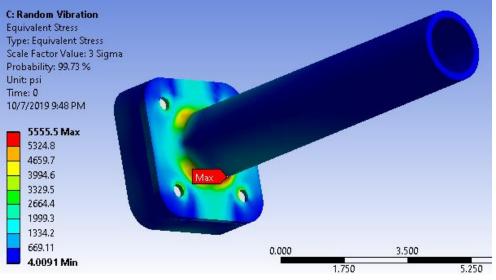
Acceleration & Stress Response to Random Vibration





Max. Acc. PSD Response = $9.08 \text{ G}^2/\text{Hz}$

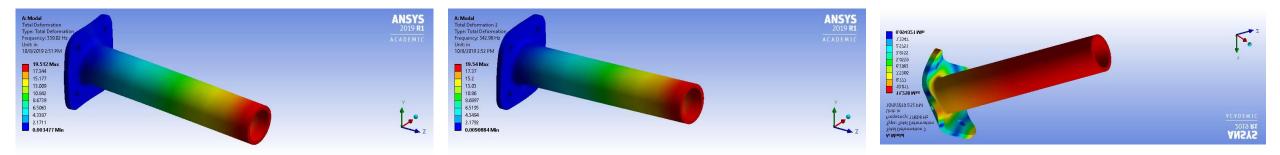


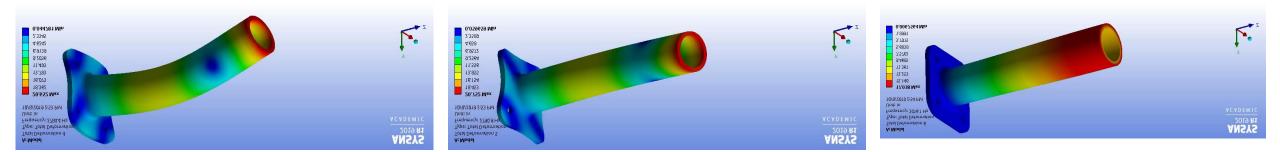


Max. Equiv. Stress = 5.6 KSI



Modal Analysis of Component with Remote Points

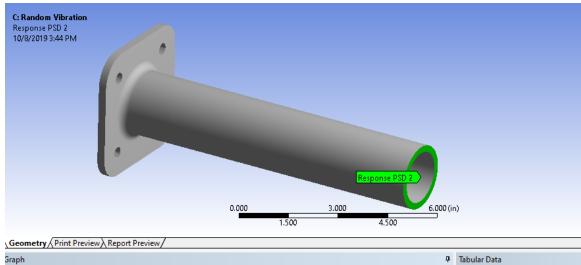


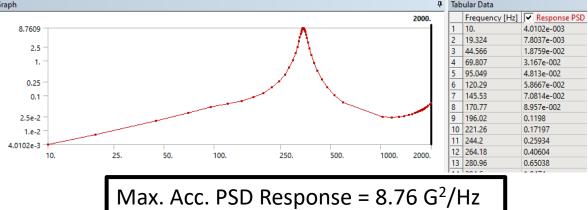


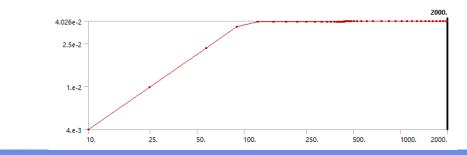
First Six Modes with Remote Points

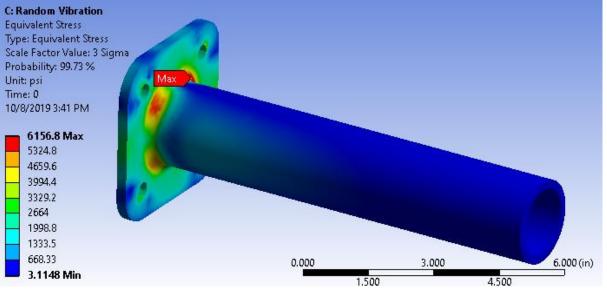


Acceleration & Stress Response to Random Vibration









Max. Equiv. Stress = 6.2 KSI



Conclusions

- Fixing bolt holes leads to unconservative stress response
- Explicit modelling of bolts, while more laborious, produces higher and more realistic stress estimates.
- Remote point approximation will predict higher stresses and lower accelerations
- Fixing bolt holes may be acceptable for low excitation levels as the bolts will remain tightened.
- For higher excitation level explicit bolt modelling is recommended.